

σ_c
 $\sigma_{cm} \pm k_t f_{ct}$
 $\epsilon_{cm} = k_t \frac{f_{ct}}{E_c}$

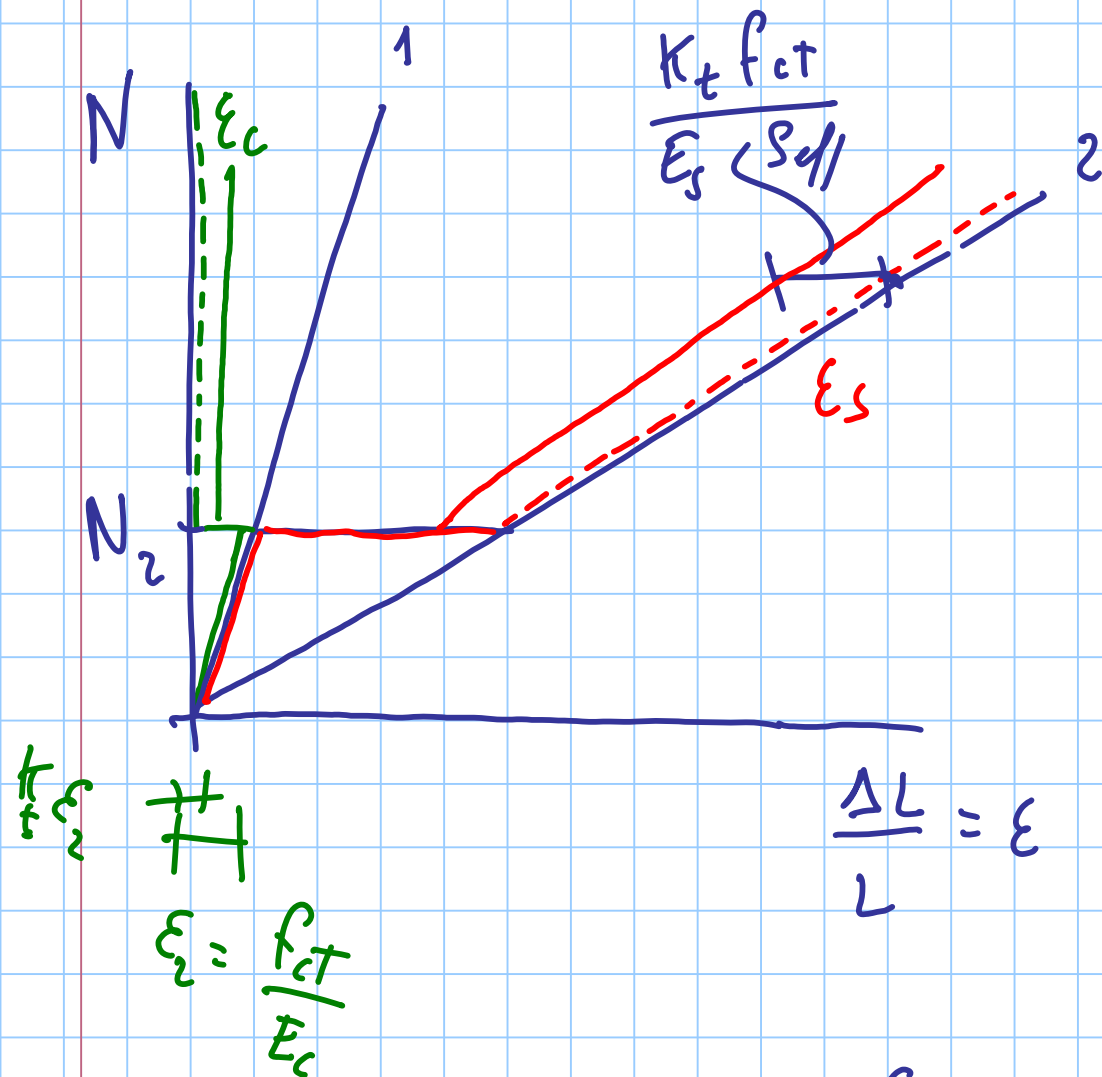
The diagram shows a linear stress distribution across the height of the beam. The top horizontal line is blue, and the bottom horizontal line is blue. A pink line represents the average stress σ_{cm} . The stress distribution is shown as a blue zigzag line. The top of the beam is under tension, and the bottom is under compression.

σ_{s2}
 σ_{sm}
 ϵ_{sm}

The diagram shows a linear stress distribution across the height of the beam. The top horizontal line is dashed blue, and the bottom horizontal line is solid blue. A pink line represents the average stress σ_{sm} . The stress distribution is shown as a blue zigzag line. The top of the beam is under tension, and the bottom is under compression.

σ_s
 $\epsilon_{sm} = \epsilon_{s2} - \frac{k_t f_{ct}}{E_s \rho_f}$

The diagram shows a linear stress distribution across the height of the beam. The top horizontal line is dashed blue, and the bottom horizontal line is solid blue. A pink line represents the average stress σ_{sm} . The stress distribution is shown as a blue zigzag line. The top of the beam is under tension, and the bottom is under compression.



— cls

— acc

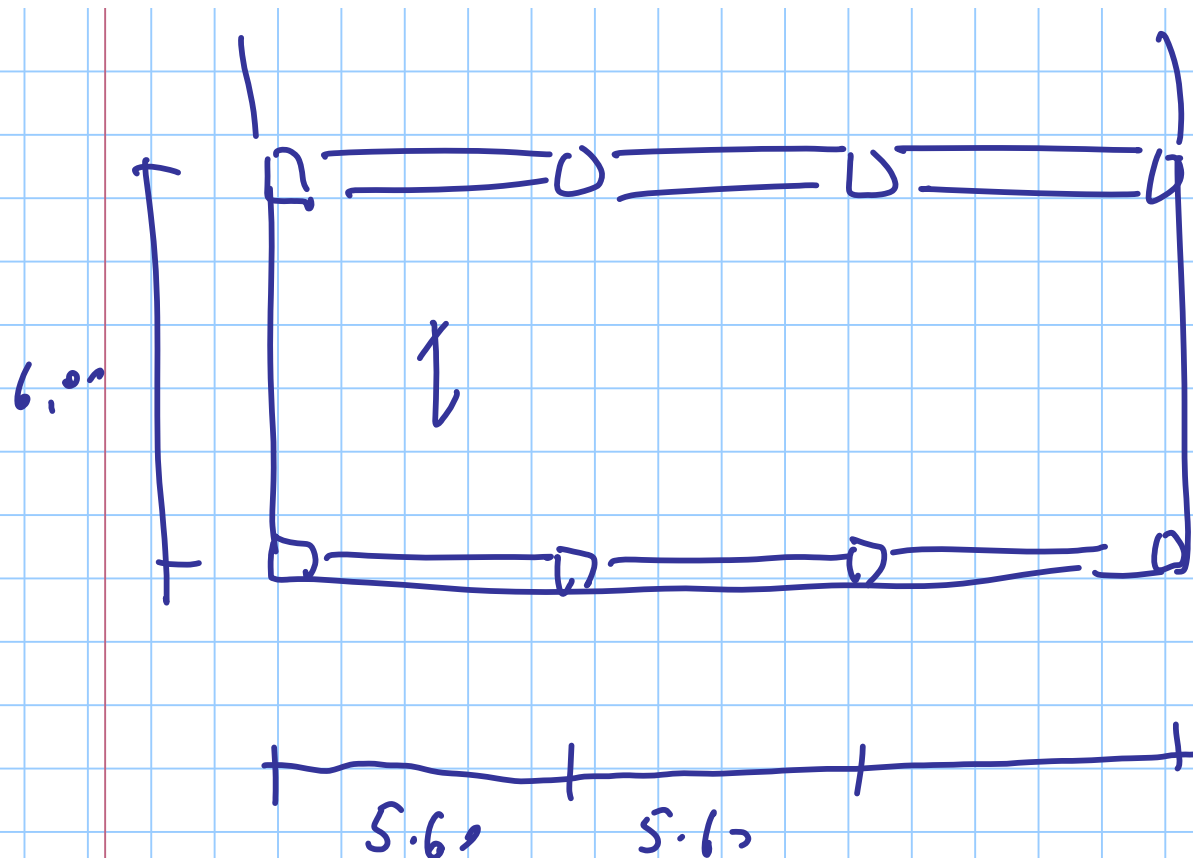
linee tra due punti etc
sotto da 1' e 2'

linee continue
TENSION STIFFENING

$$W_k = S_{zmax} (\epsilon_{sm} - \epsilon_{cm})$$

\downarrow
 dist. fence

Condizioni ambientali	Combinazione di carico	
	frequente	quasi permanente
Ordinarie: classe X0, XC1, XC2, XC3, XF1	$w_k \leq 0.4 \text{ mm}$	$w_k \leq 0.3 \text{ mm}$
Aggressive: classe XC4, XD1, XS1, XA1, XA2, XF2, XF3	$w_k \leq 0.3 \text{ mm}$	$w_k \leq 0.2 \text{ mm}$
Molto aggressive: classe XD2, XD3, XS2, XS3, XA3, XF4	$w_k \leq 0.2 \text{ mm}$	$w_k \leq 0.2 \text{ mm}$



CARICHI SUI TRAVE

		q_k	q_k
solaio,	3.0 m	12.0	12.0
Trave		3.5	
perimet.		<u>1.2</u>	<u>12.0</u>
		16.7	

solaio

$$q_k = 4.0 \text{ kN/m}^2$$

$$q_k = 4.0 \text{ kN/m}^2$$

p.p. Trave

$$3.0 \times 6.0 \quad q_k = 3.5 \text{ kN/m}$$

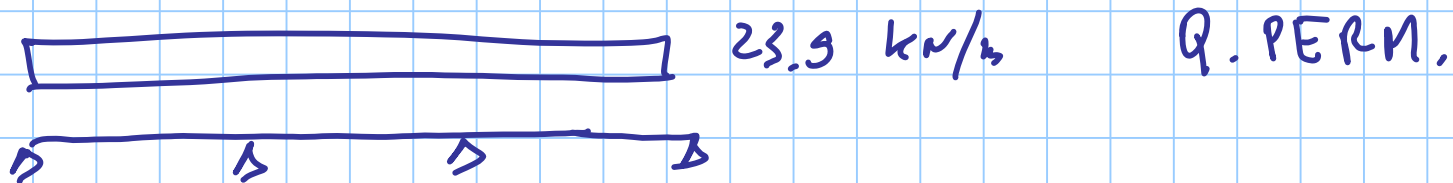
perimet.

$$q_k = 1.2 \text{ kN/m}$$

$$\psi_1 = 0.7$$

$$\psi_2 = 0.6$$

				Tot
frecuente	16.7	$12.0 \times 0.7 = 8.4$	kN/m	25.1
q. perm.	16.7	$12.0 \times 0.6 = 7.2$	kN/m	23.9



A diagram shows a beam with a triangular load. The load is represented by a solid line with peaks and valleys. A dashed line indicates the deflection curve. The maximum deflection is labeled "52" (highlighted in green) and "9l^2/16". The calculation for the maximum deflection is shown as follows:

$$\frac{9/2 l^2}{8} = \frac{9 l^2}{16} = \frac{23.9 \times 5.60^2}{16} = 46.8 \text{ kNm}$$

all. SLV

$$16.7 \times 1.3$$

$$12.0 \times 1.5$$

$$\text{Tot } 39.7 \text{ kN/m}$$

$$21.7$$

$$18$$

$$M_{rd} = \frac{ql^2}{16} = 77.8$$

$$M_{rd, \text{max}} = 90 \text{ kNm}$$

basterebbero

3 $\phi 14$

$$A_s = \frac{90 \times 10}{0.9 \times 0.56 \times 391.3} = 4.6 \text{ cm}^2$$

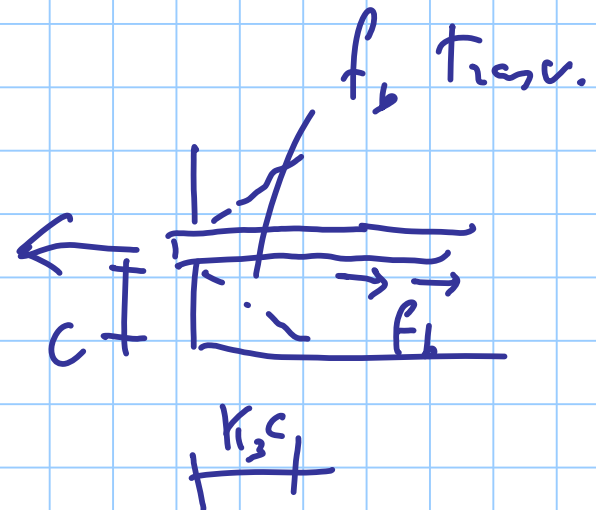
il progettista ne ha messi 4 $\phi 14$

$$S_{2, \max} = \overbrace{K_3 C}^{102} + \overbrace{K_1 K_2 K_4}^{116.1} \frac{\phi}{S_{eff}} = 218.1 \text{ m}_h$$

$\begin{matrix} 3.4 & 30 \\ / & / \end{matrix}$
 $\begin{matrix} 0.8 & 0.5 & 0.425 \\ / & / & / \end{matrix}$
 $\begin{matrix} 14 \\ / \end{matrix}$

$$A_{eff} = 2.5 \times 4 \times 30 = 300 \text{ cm}^2$$

$$S_{eff} = \frac{4 \times 1.54}{300} = 0.0205$$



nota: π averei men. $2\phi 20$ anziché $4\phi 14$

$$\rho_{d1} = \frac{2 \times 3.14}{300} = 0.0209$$

$$K_1 K_2 K_4 \frac{\phi}{\rho_{d1}} = 162.7$$

$$S_{2max} = 264.7 \text{ mm}$$

se i hanno diametri diversi:

$$n_1 \phi_1$$

$$n_2 \phi_2$$

$$\phi_{eq} = \frac{n_1 \phi_1^2 + n_2 \phi_2^2}{n_1 \phi_1 + n_2 \phi_2}$$

$$\frac{2 \times 14^2 + 1 \times 20^2}{2 \times 14 + 1 \times 20} = 16.5$$

es. $2\phi 14 + 1\phi 20$

$$\epsilon_{sm} - \epsilon_{cm} - \left(\epsilon_{s2} - \frac{K_t f_{ct}}{E_s \rho_d} - \frac{K_t f_{ct}}{E_c} \right) =$$

$$= \epsilon_{s2} - \frac{K_t f_{ct}}{E_s \rho_d} \left(1 + \frac{E_s \rho_d}{E_c} \right)$$

$$f_{ct} \rightarrow f_{ctm}$$

$$c 25/30 \quad f_{ctm} = 2.56 \text{ MPa}$$

$$K_t = 0.6$$

$$0.4 \quad \left(\frac{\text{large}}{\text{small } E_c} \right)$$

ξ_{s2} rigidez: verifique 2° taxa.

$$n = 15$$

taxa x

$$\sigma_s = n \frac{M}{I} y$$

$$\epsilon_s = \frac{\sigma_s}{E_s}$$

$$A_s = \frac{M}{0.9 A \sigma_s} \Rightarrow$$

$$\sigma_s = \frac{M}{0.9 A A_s}$$

Q. PERM,

$$M = 52.0 \text{ kNm}$$

$$\sigma_s = \frac{52.0 \times 10}{0.9 \times 0.56 \times 6.16}$$

$$\sigma_{s2} = 167.5 \text{ MPa}$$

$$\epsilon_{s2} = \frac{167.5}{200000} = 0.000837$$

$$0.837 \times 10^{-3}$$

$$\epsilon_{sm} - \epsilon_{cm} = \epsilon_{s2} - \frac{K_t f_{ct}}{E_s \rho_{41}} \left(1 + \left| \frac{E_s}{E_c} \right| \rho_{41} \right)$$

Annotations for the first term: 0.4 (above K_t), 2.56 (above f_{ct}), 200000 (below E_s), 0.0205 (below ρ_{41}).
 A bracket under these four values indicates 0.250×10^{-3} .

Annotations for the second term: 1.13 (above the bracket), 6.35 (below E_c), 0.0205 (below ρ_{41}).

$$= 0.837 \times 10^{-3} - 0.250 \times 10^{-3} \times 1.13 = 0.555 \times 10^{-3}$$

$$W_k = S_{2.5\%} (\epsilon_{sm} - \epsilon_{cm}) =$$

$$= 218.1 \times 0.555 \times 10^{-3} = 0.121 \text{ mm}$$

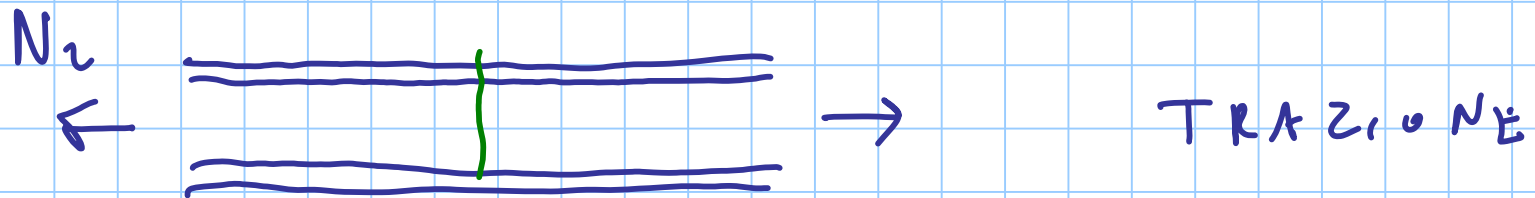
$$W_k < 0.2 \text{ mm} \quad \text{OK}$$

per limitare le fessure

— evitare diametri elevati

— evitare eccessive distanze tra le barre

mettere armature tese non troppo basse (come quantità)



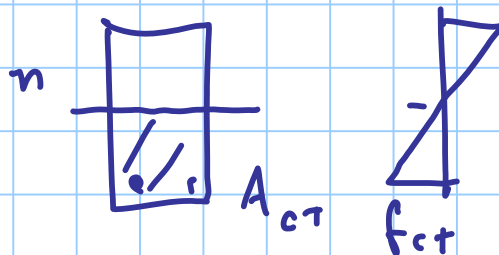
N_2 cl_s $N = A_{ct} f_{ct}$

$A_s f_y > K A_{ct} f_{ct}$

$K = 1$ Trazione

$K = 0.4$ flessione

FLESSIONE



cl_s $N = A_{ct} \frac{f_{ct}}{2}$

$$A_s \geq 0.26 \frac{f_{ct}}{f_{yh}} b d \quad \text{flexion}$$

C 25/30

$$A_s \geq 0.00148 b d \approx 0.0015 b d$$

B 450 C

30 x 50

$$A_s \geq 0.00148 \times 30 \times 46 = 2.04 \text{ cm}^2$$

ϕ

Tensione nell'acciaio [MPa]	Diametro massimo delle barre [mm]		
	$w_k \leq 0.4 \text{ mm}$	$w_k \leq 0.3 \text{ mm}$	$w_k \leq 0.2 \text{ mm}$
160	40	32	25
200	32	25	16
240	20	16	12
280	16	12	8
320	12	10	6
360	10	8	5
400	8	6	4
450	6	5	—

Nota: i valori della tabella sono ricavati assumendo $f_{ctm}=2.9 \text{ MPa}$, $h_{cr}=0.5 h$, $c=0.1 h$, $k_c=0.4$.

$$\phi_s = \phi_s^* \frac{f_{ctm}}{2.9} \frac{k_c}{2} \frac{h_{cr}}{c}$$

nel caso di flessione

$$\phi_s = \phi_s^* \frac{f_{ctm}}{2.9} \frac{h_{cr}}{8c}$$

nel caso di trazione

$$\phi = 25 \frac{2.56}{2.9} \frac{0.4}{2} \frac{30}{4} = 33$$

Tensione nell'acciaio [MPa]	Spaziatura massima delle barre [mm]		
	$w_k \leq 0.4 \text{ mm}$	$w_k \leq 0.3 \text{ mm}$	$w_k \leq 0.2 \text{ mm}$
160	300	300	200
200	300	250	150
240	250	200	100
280	200	150	50
320	150	100	—
360	100	50	—

DEFORMAZIONI

Tab. 5. Valori limite del rapporto l/h (calcestruzzo C25/30, acciaio B450C)

Sistema strutturale	k	ρ 0.5%	ρ 1.0%	ρ 1.5%
Travi semplicemente appoggiate, piastre semplicemente appoggiate mono o bidirezionali	1.0	20.6	16.4	15.0
Campata terminale di trave continua o piastre continue monodirezionali o piastre bidirezionali continue su un lato lungo	1.3	26.7	21.3	19.5
Campata intermedia di travi o di piastre mono o bidirezionali	1.5	30.8	24.6	22.5
Piastre sorrette da pilastri senza travi (con riferimento alla luce maggiore)	1.2	24.7	19.7	18.0
Mensole	0.4	8.2	6.6	6.0

$$\rho = \frac{A_s}{b d}$$

$$\frac{l}{h} \leq k \left(11 + \frac{0.0015 f_{ck}}{\rho + \rho'} \right) \frac{500}{f_{yk}} \frac{A_{s,eff}}{A_{s,calc}}$$

$$\rho = 0.78 \%$$

$$0.4 \left(11 + \frac{0.0015 \times 25}{0.0078} \right) \frac{500}{450} = 7.02$$

$$h = 24 \text{ m}$$

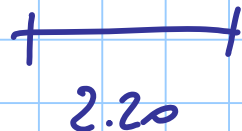


$$g_d + q_d = 10.0 \text{ kN/m}^2$$

$$S < 0$$



$$d = 21 \text{ m}$$



$$M = \frac{g l^2}{2} = 24.2 \text{ kNm}$$

$$A_s = \frac{M}{0.9 d f_{yd}} = 1.64 \text{ cm}^2/\text{m} \quad \text{a T2.5 cm}$$

2 T2.5 cm

$$b = 20 \text{ cm}$$

$$\rho = \frac{A_s}{b d}$$

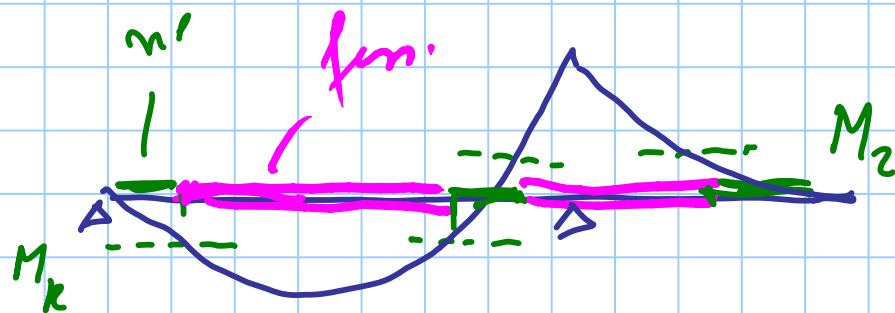
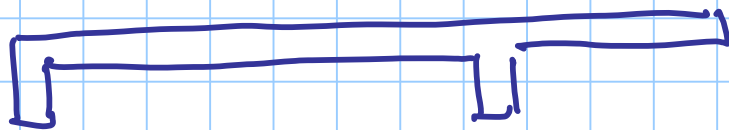
$$\rho = \frac{1.64}{10 \times 21} = 0.0078$$

$$0.78 \%$$

$$\frac{l}{h} = \frac{220}{24} = 9.16 > 7.02$$

Non So D, SFATTA

CALCOLO DELLA FRECCIA



per il calc. b

E I

M_2 mom. flessione

modul elastico

E_c

nel tempo

$$\frac{E_c}{1 + \phi}$$

↑
coeff. viscosità

I

2° modell di comport.

ma dove $M < M_2$ 1° modell

I varia anche per

- variazione momento
- cambio segno M

is it Tension stiffening?

mod. simplify & calculations

T_{cr} , I_{min} & use guesst

sugg. notation

- use I_2 (2nd r.d.) $\rightarrow \delta_2$
- use I_1 (1st r.d.) $\rightarrow \delta_1$

$$\delta = \zeta \delta_2 + (1 - \zeta) \delta_1$$

$$\zeta = 1 - \beta \left(\frac{M_2}{M} \right)^2$$

$$\beta = 0.5$$

carichi per man.

oppure dividere in conci, calcolare le rigidità

tra EI_1 e EI_2 con ζ

integrare le curvature

TENSIONI IN ESERCIZIO

— effetti viscosi m.m. forti a σ alte

comb. quasi perm. $\sigma_c \leq 0.45 f_{ck}$

— fessurazioni per compressioni

comb. rare

$$\sigma_c \leq 0.6 f_{ck}$$

— fessurazione per σ_s elevate

$$\sigma_s \leq 0.8 f_{yk}$$

tensioni in esercizio

2° modello di comportamento