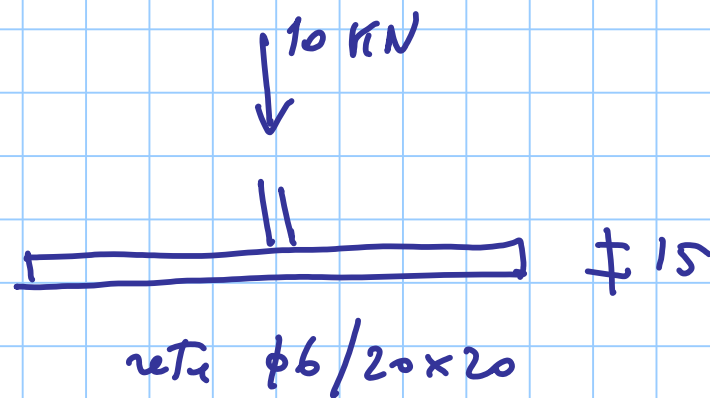
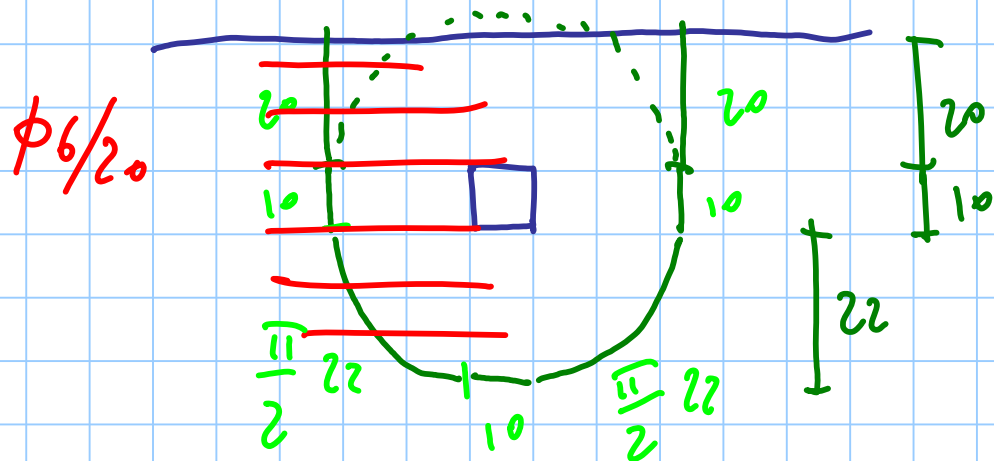


PIANTA



SEZ

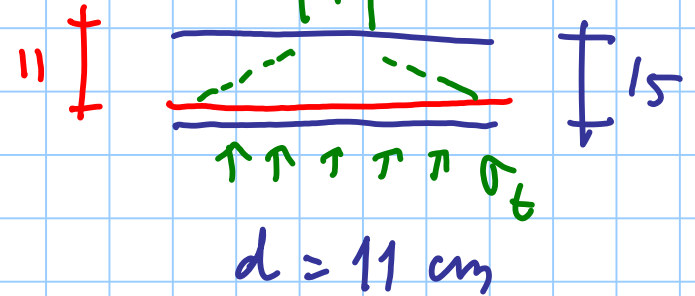
PIANTA



$$n = 139 \text{ cm}$$

$$\tau_{Ed} = \frac{V_{Ed}}{n d} = \frac{10 \times 10^3}{139 \times 11 \times 10^2} = 0.065 \text{ MPa}$$

SEZ



$$V = F - A \sigma_c$$

$$\tau_{transm} \sigma_t$$

$$v_{R1,c} = 0.18 K \sqrt[3]{100 \rho_e f_{cr}}$$

$\gamma_c$

le piñ grande

$$0.035 \sqrt{K^3 \rho_{cr}}$$

$$K = 1 + \sqrt{\frac{200}{\lambda}} \rightarrow K = 2$$

$$d = 110 \text{ mm}$$

$$\rho_e = \sqrt{\rho_{e,x} \cdot \rho_{e,y}}$$

$$\rho_{e,x} = \rho_{e,y} = \frac{A(\phi 6)}{20 \times 11} = \frac{0.28}{220} = 0.00127$$

$$0.18 \times \frac{\sqrt[3]{100 \rho_e f_{cr}}}{\gamma_c} = 0.18 \times 2 \times \frac{\sqrt[3]{100 \times 0.00127 \times 20}}{1.5} = 0.327$$

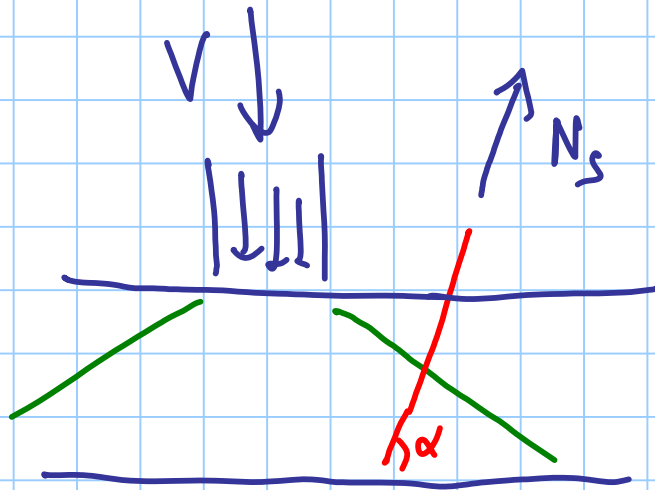
$$0.035 \sqrt{k^3 f_{cr}} = 0.035 \sqrt{2^3 \times 20} = 0.443$$

$$\sigma_{R1,c} = 0.443 \text{ MPa}$$

$$\sigma_{Ed} = 0.065 \text{ MPa} < \sigma_{R1,c} \quad \text{no occurrence of cracking}$$

$$V_{rd} = 0.75 V_{rd,c} + \text{conf. armature } V_{wd}$$

(VEDI MODELLO "NORMALE")



$$N_s = A_s f_{yd}$$

$$A_s f_{yd} \sin \alpha$$

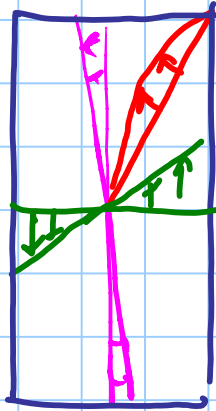
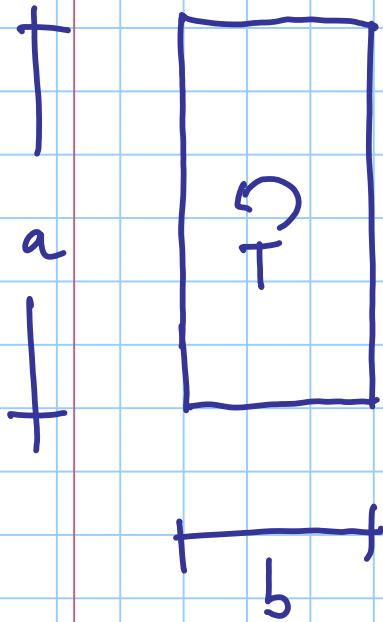
$$\alpha \geq 45^\circ$$

$$\alpha \geq 50^\circ$$

$$V_{wd} = A_{s, \text{tot}} f_{yd} \sin \alpha$$

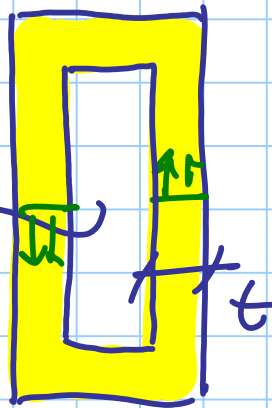
tutta l'armatura che attraversa perim critico

# TORSIONE



$$\gamma_{max} = \gamma \frac{T}{a b^2}$$

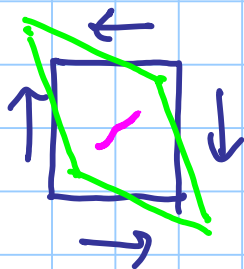
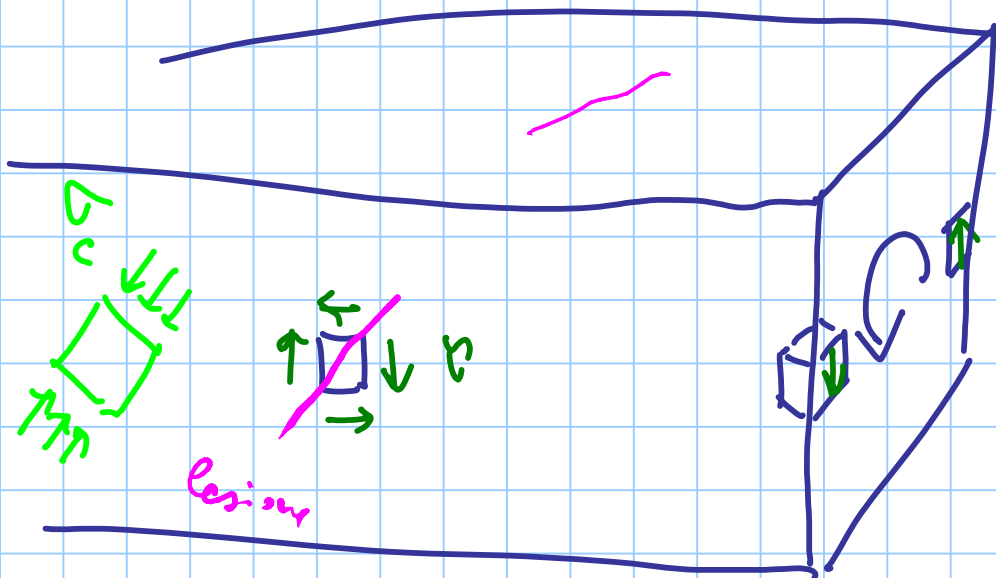
contributo  
Transversale

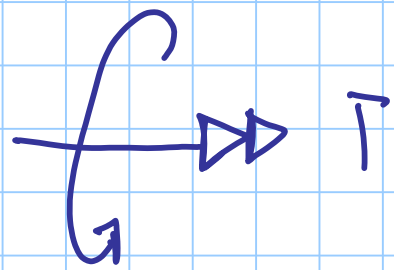
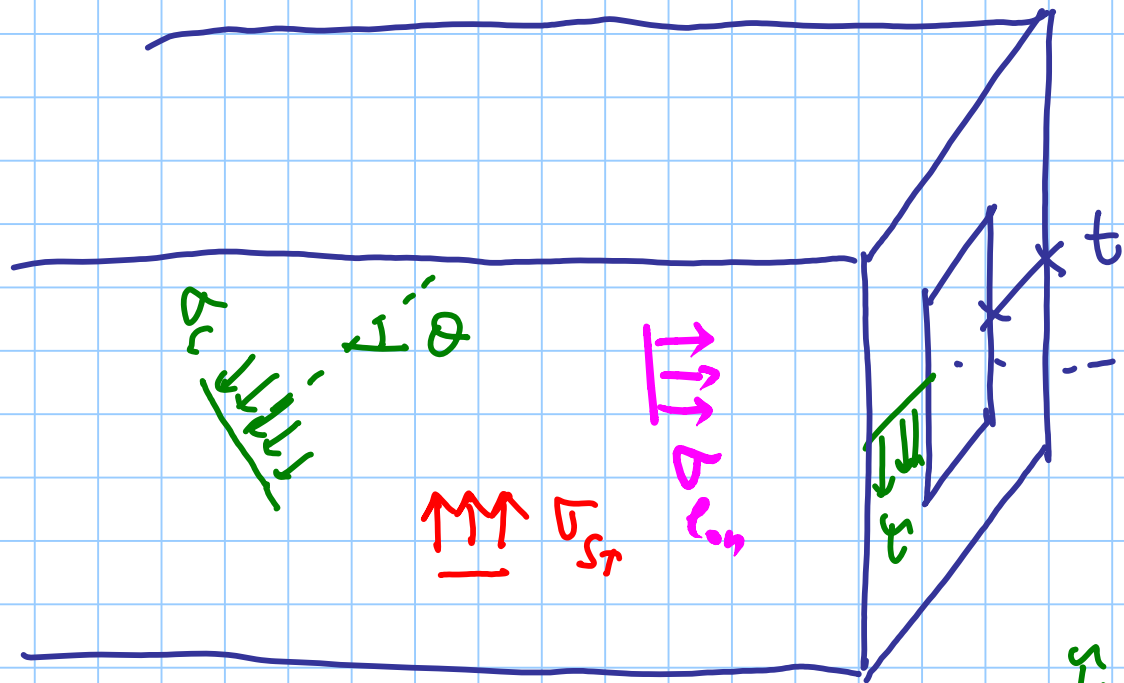


la torsione è portata

da una sezione "scatolare"

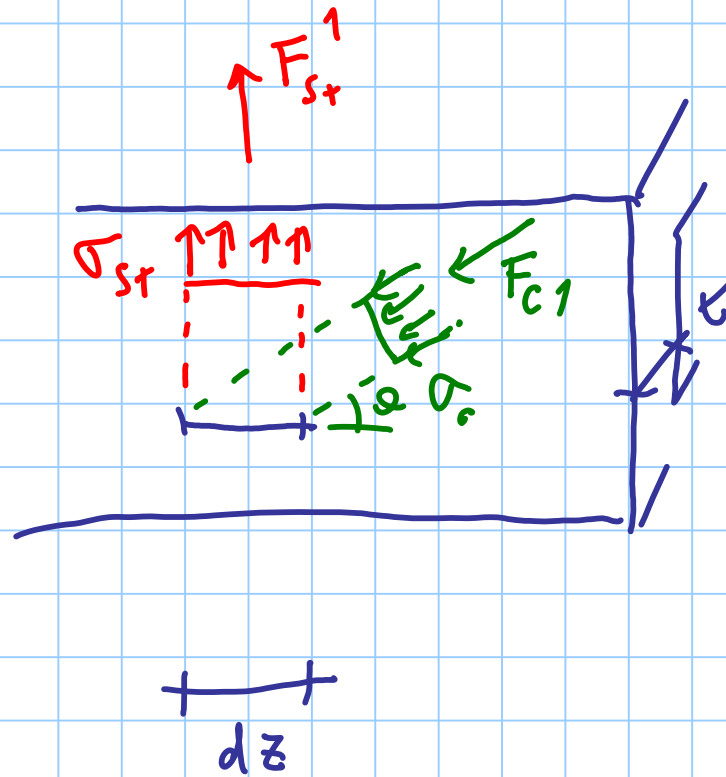
$$\gamma = \frac{T}{2 A_k t}$$





$$\tau = \frac{T}{2 A_k t}$$





$$F_{sr}' = \frac{A_{sr}}{s} dz \sigma_{sr}$$

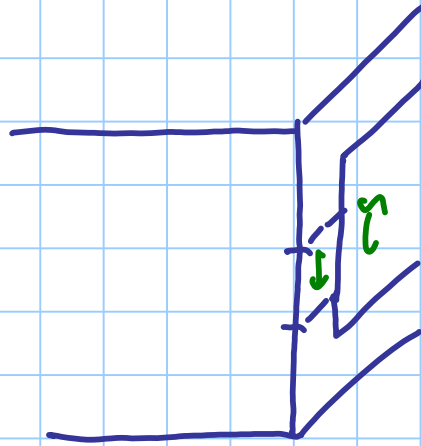
$$F_c' = dz \sin \theta \tau$$

$$F_c' \sin \theta$$

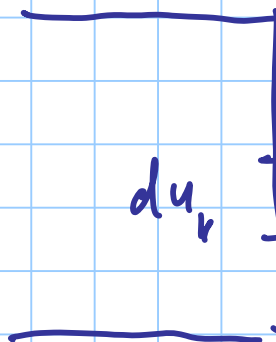
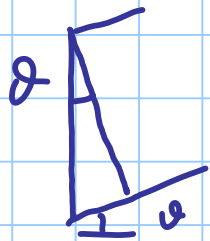
equil. Tension vertical

$$\frac{A_{sr}}{s} dz \sigma_{sr} = dz \sin^2 \theta \tau$$

$$du_y \uparrow$$

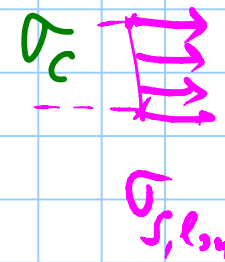


$$\gamma = \frac{T}{2 A_k t}$$



$$du_r$$

$$N_c^2 \sin \theta = t \cdot du_y$$



$$= \frac{T}{2 A_k} du_k$$

$$N_c^2 = du_k \cos \theta \cdot t \cdot \sigma_c$$

$$\cancel{du_k} \cos \theta \leq \sigma_c \sin \theta = \frac{T}{2 A_k} \cancel{du_k}$$

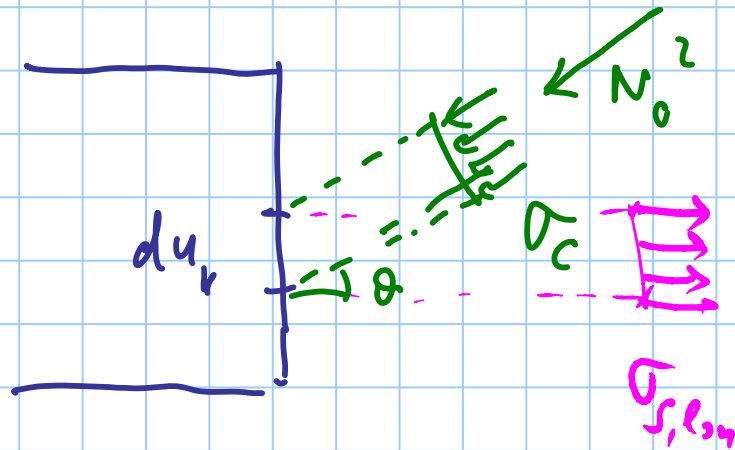
$$\sigma_c = \frac{T}{2 A_k \sin \theta \cos \theta} \leq \nu f_{cl}$$

$$\begin{aligned} T_{Rd, max} &= 2 A_k \leq \nu f_{cl} \sin \theta \cos \theta = \\ &= 2 A_k \leq \nu f_{cl} \frac{\cos 2\theta}{1 + \cos^2 \theta} \end{aligned}$$

$$\sigma_{st} = \frac{\sin^2 \theta \cdot t \cdot \sigma_c}{\frac{A_{st}}{s}} = \frac{\cancel{\sin^2 \theta} \cdot \cancel{t} \cdot T}{2 A_k \cdot \cancel{t} \cdot \cancel{\sin \theta} \cdot \cos \theta} = \frac{T}{2 A_k \cos \theta \cdot \frac{A_{st}}{s}}$$

$$= \frac{T}{2 A_k \cos \theta \cdot \frac{A_{st}}{s}} \leq f_{yt}$$

$$T_{Rl, st} = 2 A_k \cdot \frac{A_{st}}{s} \cdot f_{yt} \cdot \cot \theta$$



$$N_0^2 \cos \theta =$$

$$= \frac{A_{s,lon}}{n} du_r \sigma_{s,lon}$$

$$\cancel{du_r} \cos^2 \theta \cdot t \cdot \sigma_c = \frac{A_{s,lon}}{n} \cancel{du_r} \sigma_{s,lon}$$

$$\sigma_{s,lon} = \frac{\cos^2 \theta \cdot t \cdot \sigma_c}{\frac{A_{s,lon}}{n}} = \frac{\cancel{\cos^2 \theta} \cdot \cancel{t} \cdot \overbrace{2 A_n}^T \cdot \cancel{\sin \theta \cos \theta}}{\frac{A_{s,lon}}{n}}$$

$$\sigma_{s,ln} = \frac{T \cot \theta}{2 A_K \frac{A_{s,ln}}{n}} \leq f_{yd}$$

$$T_{Rd,ln} = 2 A_K \frac{A_{s,ln}}{n} f_{yd} \frac{1}{\cot \theta}$$

$$T_{Rd,max} = \underbrace{2 A_k}_{\text{pink}} \underbrace{t}_{\text{green}} \nu f_{cd} \frac{\cot \theta}{1 + \cot^2 \theta}$$

$$T_{Rd,st} = \underbrace{2 A_k}_{\text{pink}} \frac{A_{st}}{s} f_{yd} \cot \theta$$

$$T_{Rd,bn} = \underbrace{2 A_k}_{\text{pink}} \underbrace{\frac{A_{s,bn}}{s}}_{\text{green}} f_{yd} \frac{1}{\cot \theta}$$

$$V_{Rd,max} = \underbrace{z}_{\text{pink}} \underbrace{b}_{\text{green}} \nu f_{cd} \frac{\cot \theta}{1 + \cot^2 \theta}$$

$$V_{Rd,s} = \underbrace{z}_{\text{pink}} \frac{A_{st}}{s} f_{yd} \cot \theta$$

$$V_{Rd,pm} = \underbrace{z}_{\text{pink}} \underbrace{\frac{A_{pm}}{z}}_{\text{green}} f_{yd} \frac{1}{\cot \theta (z)}$$

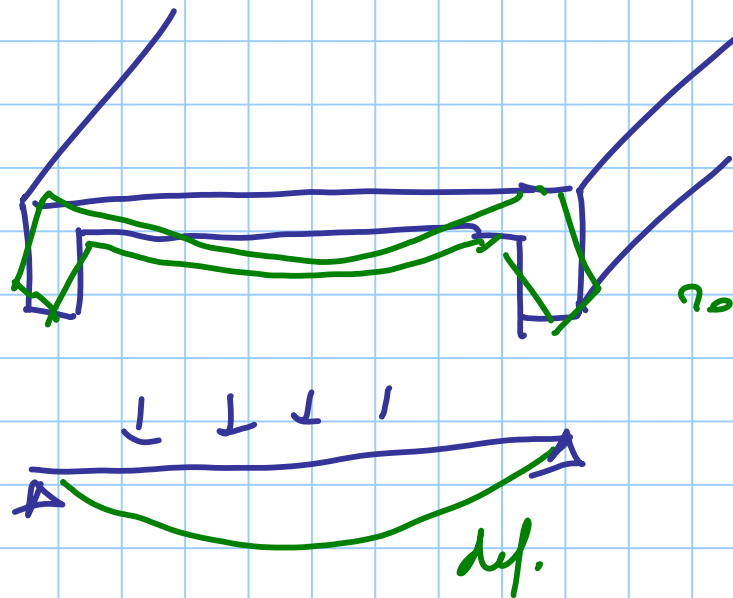
TORSIONE

PER CONGRUENZA

SI TRASCURA

~

PER EQUILIBRIO



rotazione torsionale della  $T_{re}$ ,  
per congruenza



$$t = \frac{A}{u} \geq 2c$$

Exmp.

$$30 \times 50$$

$$A = 1500 \text{ cm}^2$$

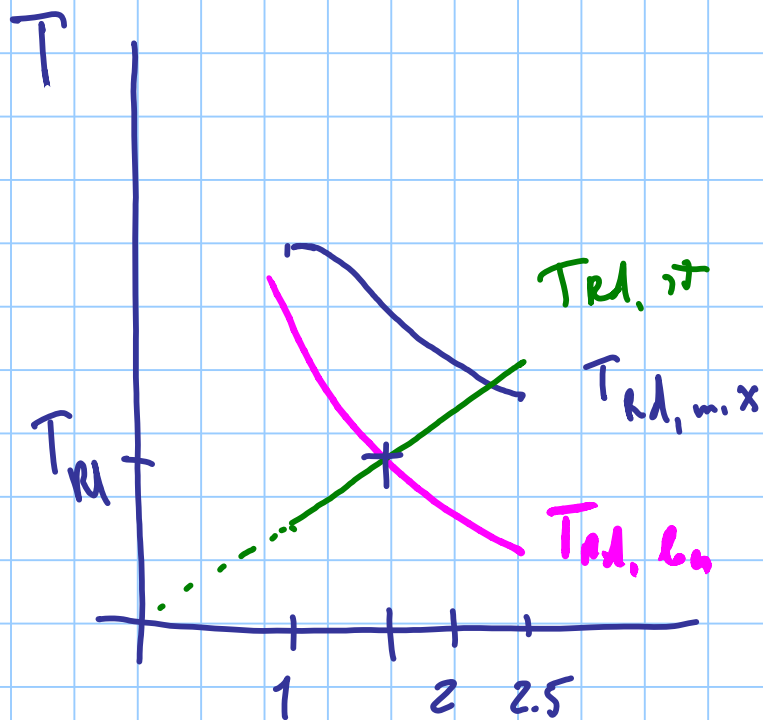
$$u = 160 \text{ cm}$$

$$c = 4 \text{ cm}$$

$$t = \frac{1500}{160} = 9.4 \text{ cm}$$

$$x \ c = 5 \text{ cm}$$

$$t = 10 \text{ cm}$$

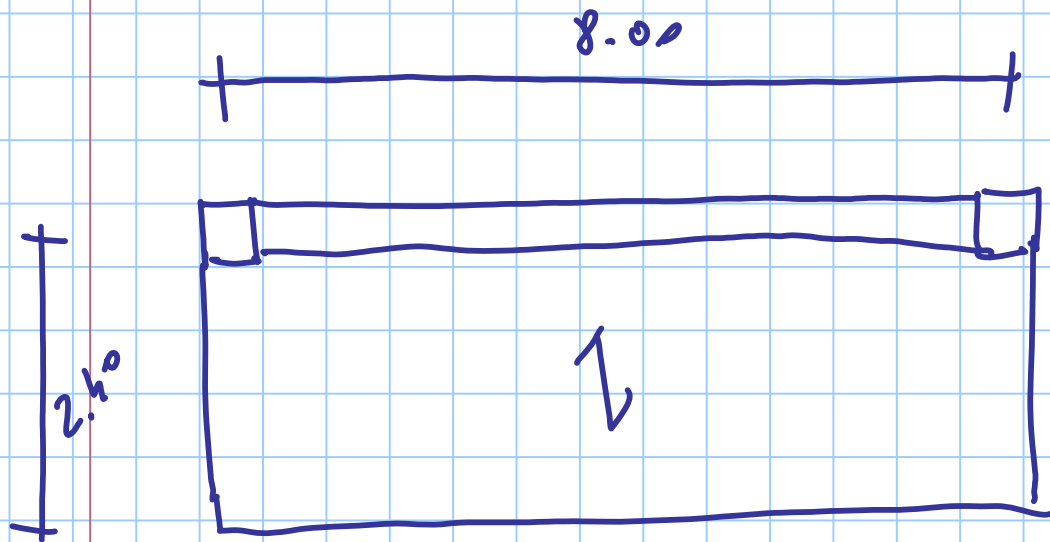


$$T_{Rd} = \text{MIN} \left( T_{Rd,mix} ; T_{Rd,ir} ; T_{Rd,bn} \right)$$

$$T_{Rd,mix} = \left[ \frac{\cot \theta}{1 + \cot^2 \theta} \right]$$

$$T_{Rd,ir} = \left[ \cot \theta \right]$$

$$T_{Rd,bn} = \left[ \frac{1}{\cot \theta} \right]$$



PLANT A

soln:-

$$g_k = 3.0 \text{ kN/m}^2$$

$$q_k = 1.0 \text{ kN/m}^2$$

$$g_d + q_d = 5.4 \text{ kN/m}^2$$

Trve

$$g_k = 3 \text{ kN/m}$$

$$g_d = 3.9 \text{ kN/m}$$

carico su Trave - vertical

car. torce  $T_1$

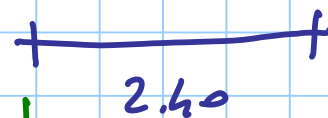
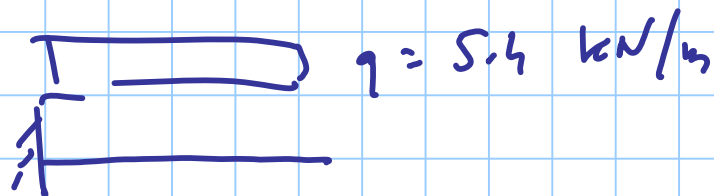
no. k: 2.40  $g_1 + g_4$   
13.0

p.p. Trave  $3.9$

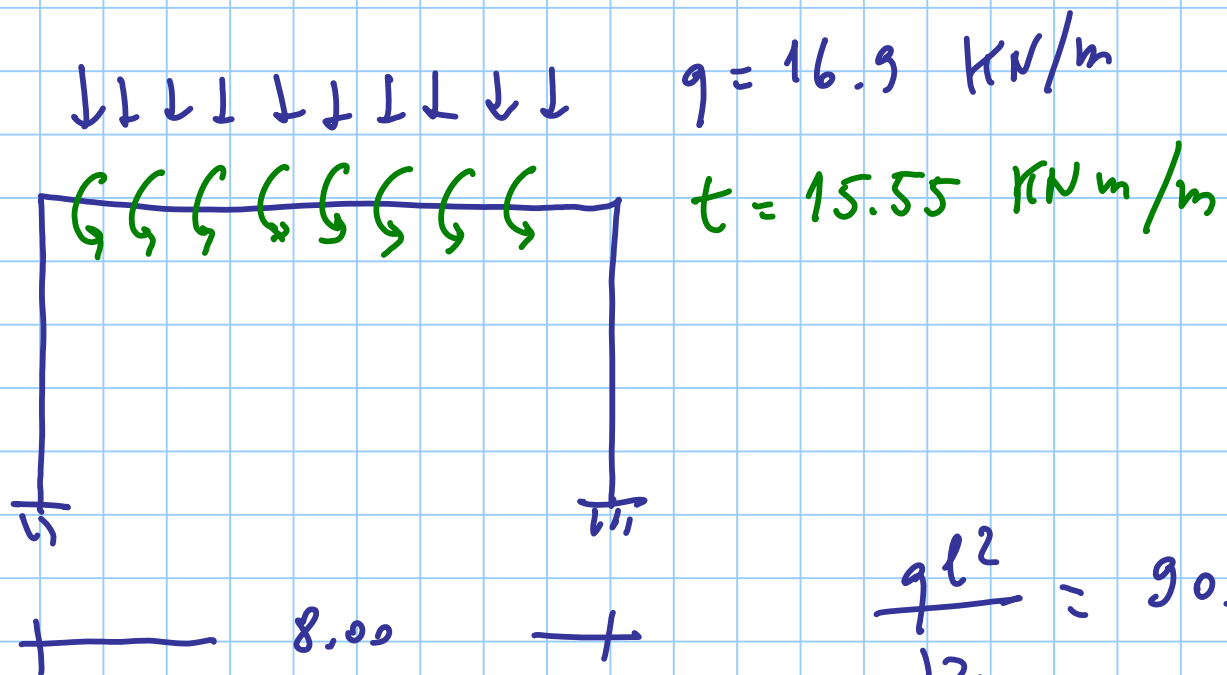
$$q = 16.9 \text{ kN/m}$$

$$t = 15.55 \text{ kNm/m}$$

perfor.

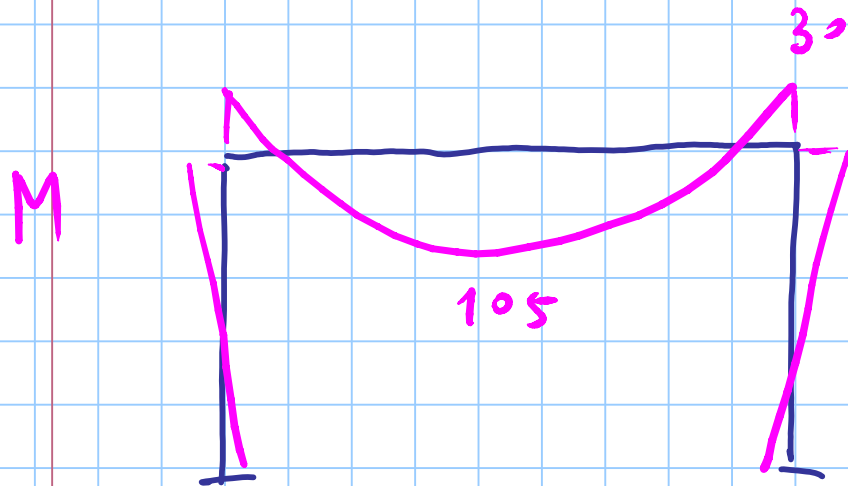


$$\downarrow q \times 2.40$$
$$\curvearrowright q \times 2.40^2 / 2 = 15.55 \text{ kNm}$$

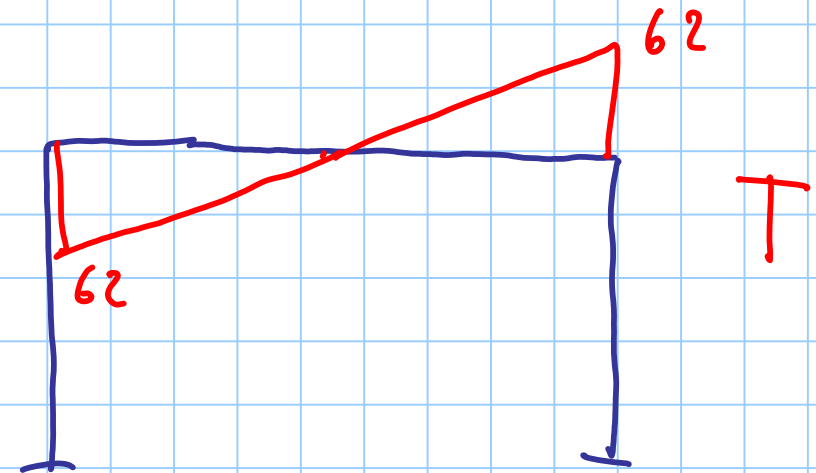


$$\frac{ql^2}{12} = 90.1 \text{ kNm}$$

$$\frac{gl^2}{8} = 135.2 \text{ kNm}$$



$$V_{max} = 67.6 \text{ kN}$$



$$T_{max} = t \frac{l}{2} = 62.2 \text{ kNm}$$

dimensionamento per torsione

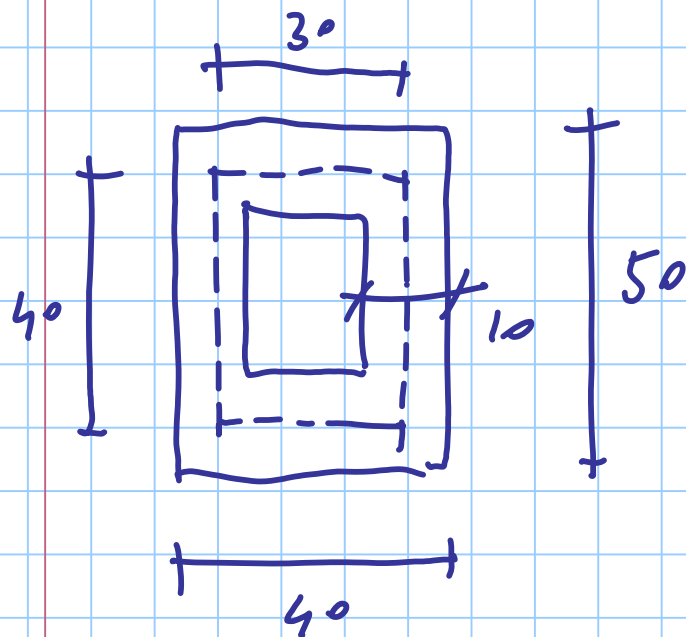
$$2 \nu f_{ct} = 14.2 \text{ MPa}$$

$$T_{Rd, \max} = 2 A_k t \nu f_{ct} \frac{\omega T d}{1 + \omega T^2 d} \cdot 0.4$$

$$A_k \cdot t = \frac{T_{Rd, \max}}{2 \nu f_{ct} \frac{\omega T d}{1 + \omega T^2 d}} = \frac{62.2 \times 10^6}{14.2 \times 0.4} \times 10^3 =$$

$$= 10950 \text{ cm}^3$$

$$\alpha t = 10 \text{ cm} \quad A_k = 1095 \text{ cm}^2 \quad \rightarrow 30 \times 40$$



$$40 \times 50$$

$$t = \frac{2000}{180} = 11.1 \text{ cm}$$

$$b_k = 40 - 11.1 = 28.9 \text{ cm}$$

$$A_k \cdot t = 28.9 \times 38.9 \times 11.1 = 12479 \text{ cm}^3 > 10950$$

$$A_k = 1124 \text{ cm}^2$$



verifiche della sezione

a flessione

$$M_{Ed} \leq M_{Rd}$$

a taglio e torsione

$$\frac{V_{Ed}}{V_{Rd, max}} + \frac{T_{Ed}}{T_{Rd, max}} \leq 1$$

da questo vediamo quale  $\cot \theta$  si può usare

$$E, \quad \cot \theta = 2$$

flessione

$$A_s = \frac{M}{0.9 d f_{yd}} \quad \begin{array}{ll} \text{maxi, 105} & \rightarrow 6.6 \text{ cm}^2 \\ \text{app. 30} & \rightarrow 1.3 \text{ cm}^2 \end{array}$$

taglio e Tensione

STAFFE

$$A_{sx}^v = \frac{V_s}{z f_{yd} \cot \theta} = \frac{67.6 \times 100 \times 10}{40.5 \times 391.3 \times 2} = 2.1 \text{ cm}^2 / 3$$

ogni staffa reale  $\times n$  bracci

$$A_{sx}^T = \frac{T_s}{2 A_n f_{yd} \cot \theta} = \frac{62.2 \times 100 \times 10^3}{2 \times 1124 \times 391.3 \times 2} = 3.5 \text{ cm}^2$$

$$\text{stoffe} \text{ in} \text{ Anz. von } \frac{2.1}{2} + 3.5 = 4.55 \text{ cm}^2 / \text{m}$$

$$\phi 8 = 0.5 \text{ cm}^2 \quad 9.1 \text{ stoffe/m} \quad \phi 8 / 10$$

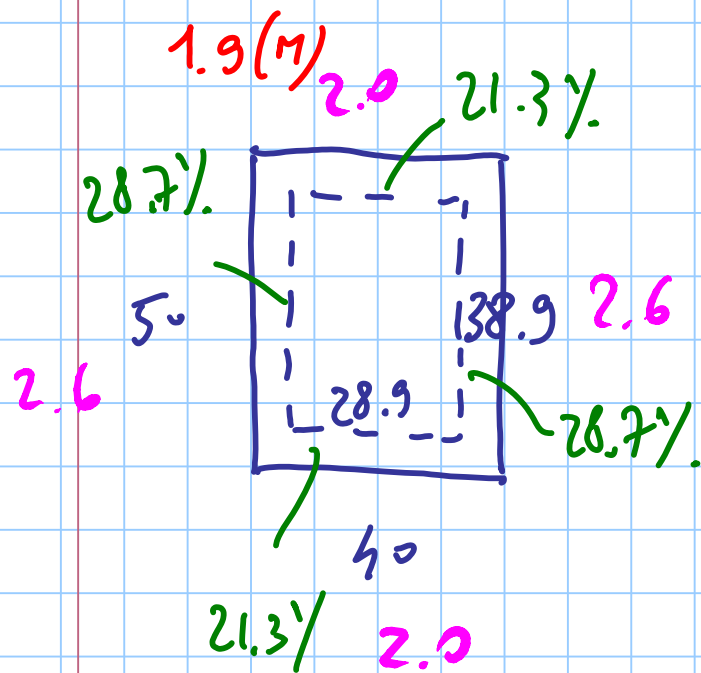
$$5 \text{ cm}^2 / \text{m}$$

püchi  $5 > 4.55$   $\text{poten. von } \cot \theta < 2$

$$[\cot \theta = 1.82]$$

armature longitudinal

$$A_{s,ln} = \frac{T n \cot \theta}{2 A_k \rho_d} = 9.2 \text{ cm}^2$$



$$n_k = 2(28.9 + 38.9) = 135.6$$

$$\frac{28.9}{135.6} = 0.213$$

$$\frac{38.9}{135.6} = 0.287$$